

## Dairy Cooling in Humid Subtropical Climates

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This guide is a climate-specific extension of the summary article “Dairy Cooling: The Benefits and Strategies,” which describes heat stress, its costs and consequences, and several strategies for its mitigation. Herein, we focus on the special challenges faced by dairy producers seeking to prevent heat stress in regions with hot and humid summers and warm winters. We discuss the various facility design options and the pros, cons and costs of the ventilation and cooling systems most commonly used in freestall barns located in these regions.

### Climate

The American Southeast (as well as parts of South America, Africa, Europe, and Asia) is dominated by a climate classified as “humid subtropical,” which is characterized by summer seasons that are both hot and humid. The high temperatures reduce the amount of heat that can be dissipated by convection, and the high humidities limit evaporation. As a consequence, dairy cows raised in these climates are, on average, more greatly affected by heat stress than cows raised in humid but cooler continental climates, not to mention cooler and drier climates.

While no system is likely to completely eliminate the effects of heat stress when both temperature and humidity levels are high, increasing air velocity over the cow and providing some form of evaporative cooling can reduce the costs associated with heat stress by over 45% (including the annualized capital costs and operating costs of the cooling systems).<sup>[1]</sup> Table 1 presents the typical number of heat-stress hours and the estimated costs of heat stress in three humid subtropical states under minimal heat stress abatement and with fans and sprinklers.

Table 1. Heat stress hours, estimated costs of heat stress under minimal heat stress abatement and with fans and sprinklers, and percent savings at three humid subtropical locations.<sup>[1]</sup>

	Typical heat stress hours <sup>1</sup>	Costs under minimal heat stress abatement <sup>2</sup> (\$/head/year)	Costs with fans and sprinklers (\$/head/year)	Percent savings (%)
Florida	4751	\$676	\$323	52
Mississippi	3810	\$580	\$318	45
Louisiana	4479	\$746	\$406	46

1. Heat-stress hours calculated from weather data in Gainesville, FL, Jackson, MS, Baton Rouge, LA when THI > 68.<sup>[2]</sup>

2. Minimal heat stress abatement assumes access to shade and good air exchange, but no additional cooling.

## Common Facilities and Ventilation Types

To better manage manure and provide improved cow comfort, many dairies in the American Southeast have adopted freestall housing. However, because winters are generally relatively warm, shade structures located in the pasture are also a viable option that is still relatively common for dry cows. For cows managed in this unconfined way, cooling ponds provide an additional option for relieving heat stress.<sup>[3]</sup>

Freestall barns in this climate are often naturally ventilated during winter and provide additional air velocity during hot weather with circulation fans or by converting to mechanical-tunnel ventilation. Barns using circulation fans for hot weather air velocity are typically more open than naturally ventilated barns in cooler climates, without endwalls or sidewalls and with wider ridge openings. Barns that are converted to tunnel ventilation during hot weather require curtain sidewalls to enclose the barn. Alternatively, some tunnel-ventilated barns have permanent sidewalls and are mechanically ventilated year around. A number of large dairies in the Midwest have built low-profile cross-ventilated (LPCV) barns in the last decade. However, LPCV barns are not common in hot and humid climates.

To protect the cows from the effects of thermal radiation, the roof of any shade structure can be painted white and insulation can be installed on its underside. White paint fades over time and insulation needs a hard surface underneath to prevent birds and rodents from nesting.

## Choosing a Ventilation and Cooling System

Because the ideal barn shape differs between naturally ventilated barns with circulation fans and mechanically ventilated barns, deciding which of these systems to implement is the first step. Additional equipment and design specifics can then be selected to provide evaporative cooling and high air velocity. This process outlined in figure 1 below.

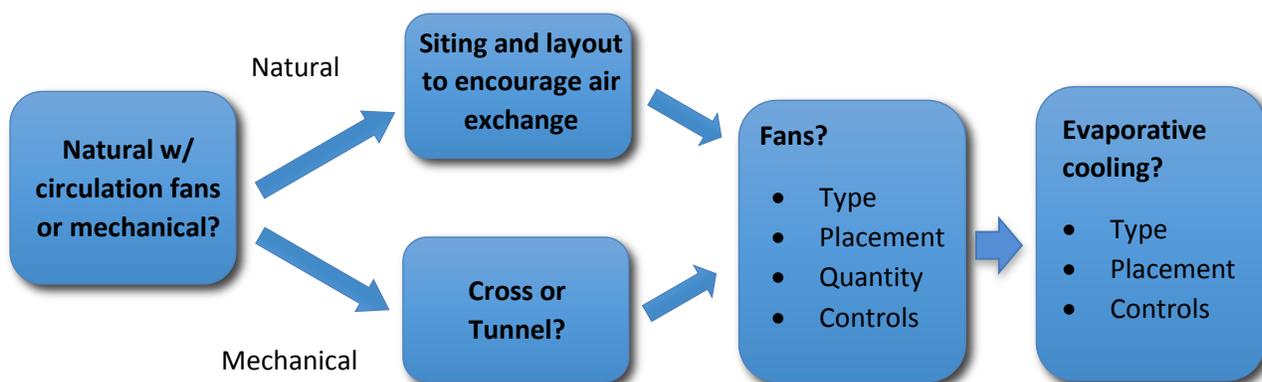


FIG. 1. Decision-making diagram outlining the steps to achieving adequate hot weather ventilation and heat stress abatement.

## Natural (with supplemental cooling) or mechanical

Both open-sided, naturally ventilated barns with supplemental cooling and totally enclosed, mechanically ventilated barns can effectively reduce heat stress in hot and humid climates, if they provide good air velocity at cow level and some form of evaporative cooling. For smaller barns, circulation fans (to supplement natural ventilation) are almost always less expensive than the exhaust fans needed for mechanical ventilation. However, the cost per cow of mechanical ventilation decreases as the tunnel-ventilated barn becomes longer (and as the cross-ventilated barn gets wider) until it approaches a “critical length,” at which point the cost of the exhaust fans used for mechanical ventilation becomes less than the cost of installing circulation fans throughout the entire barn. This critical length has been estimated to be about 500 feet for a tunnel-ventilated barn.<sup>[4]</sup> An additional cost of using mechanical ventilation is that resulting from curtain or permanent sidewalls needed to enclose the structure. The pros and cons of natural and mechanical ventilation in humid subtropical climates are summarized in Table 2.

Table 2. Pros and cons of natural and mechanical ventilation.

	Pros	Cons
Mechanical	<ul style="list-style-type: none"><li>• High controllability</li><li>• Good ventilation when no wind outside</li><li>• Can be constructed alongside other buildings, silos, and obstructions</li><li>• No flies or birds</li></ul>	<ul style="list-style-type: none"><li>• Usually higher upfront cost</li><li>• Fans run more hours per year</li><li>• Backup power needed in case of power failure</li><li>• Requires knowledge of how to operate the system</li></ul>
Natural	<ul style="list-style-type: none"><li>• Reduced operating costs</li><li>• Reduced concern due to power failure</li><li>• Reduced lighting costs</li></ul>	<ul style="list-style-type: none"><li>• Low ventilation on still summer days</li></ul>

## Naturally Ventilated Barns

### *Basic siting and layout*

Natural ventilation is effective at sites with consistent wind and away from other buildings and obstructions. Orienting the open sidewalls perpendicular to summertime prevailing winds provides the best ventilation, and an East-West orientation provides the best shade. The direction and intensity of prevailing winds can be found by consulting ‘wind rose’ diagrams, such as those found at (<http://www.wcc.nrcs.usda.gov/ftpref/downloads/climate/windrose/>). See Table 4 in the companion summary article<sup>[5]</sup> to determine the height of sidewall openings, which are generally between 10 and 16 feet high. If obstructions are present, larger sidewall openings can compensate to some extent. In Florida the recommended ridge opening is one foot plus two inches per 10 feet of building width<sup>[6]</sup>, which is wider than is typical in the Midwest or Northeast.

## **Supplemental cooling for naturally ventilated barns**

In hot and humid climates, shade and natural ventilation are not sufficient for reducing heat stress. Circulation fans and evaporative cooling should be provided in open-sided barns for effective heat stress relief.

### **Circulation fans**

#### *Type*

Other than cost, several factors that should influence the selection of circulation fans are their throw, efficiency, reliability, and ease of maintenance.

#### *Location and Quantity*

The current recommendation for circulation fan placement is to locate one row of fans over the stalls and one row over the feedline as shown in figure 2.<sup>[7]</sup> This is based on a Kansas State study that compared the performance of cows housed in pens cooled by fan-and-sprinkler systems laid out in various ways. The study found that fans placed over both the feedline and stalls significantly improved milk production, but that 2 rows of fans installed over head-to-head stalls did not provide additional benefits.<sup>[8]</sup>

However, if the fans placed above a row of head-to-head stalls are staggered—that is to say, if the first fan is placed to one side above the first stall and the next fan is placed above the alternate side of the next stall, and so on, then a more even airflow may be achieved.

Usually, circulation fans are placed 10 fan diameters apart, although some producers have opted to space fans more closely together to produce a greater air velocity. Fans should be placed a minimum of 8 feet above the barn floor<sup>[9]</sup> and tilted down at a 25- to 35-degree angle. The closer and higher the fans are mounted, the steeper the pitch. One should be able to feel the air at their feet when standing at the next fan.<sup>[10]</sup>

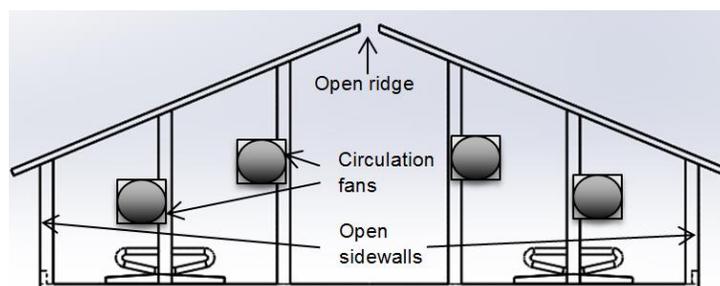


FIG. 2. Current recommendations for fan placement in a naturally ventilated freestall.<sup>[7]</sup>

See the companion summary article<sup>[5]</sup> for information on fan placement in the holding area.

### *Priority for circulation fans*

When available funds will not cover the costs of installing circulation fans in all desired locations, the funds should be applied according to the following priority:<sup>[9]</sup>

1. Holding pens
2. Close-up (dry cow) pens
3. Maternity and sick pens
4. Fresh-cow pens
5. High-producer pens
6. Low-producer pens

There is increasing evidence that dry cows should be cooled in hot and humid climates. In a study conducted at the University of Florida, a group of cows subjected to heat stress during their dry period produced an average of 11 pounds per cow per day less during the subsequent lactation than those that received cooling.<sup>[11]</sup> Another study performed at the University of Florida found the average birth weight of calves born to heat stressed mothers was 80.5 pounds, as compared to 93.7 pounds for those from mothers that had received cooling.<sup>[12]</sup>

### *Controls for circulation fans*

The most recent guidelines recommend turning circulation fans on when the temperature exceeds 68 °F.<sup>[9]</sup> However, because it is the combination of temperature and humidity that causes heat stress, the total number of hours the fans must be on per year can be reduced by including a humidity sensor and setting the fans to turn on at a THI of 68. Variable frequency drives (VFDs) also have the potential to save additional energy, but more research must be done to determine guidelines for how VFDs on circulation fans should be controlled as temperatures increase.

### ***Evaporative cooling***

A combination of evaporative cooling and circulation fans is more effective at reducing heat stress and will provide higher economic returns than will using fans alone. In fact, when the air temperature is higher than the cow's body temperature and the cow's skin is dry, fans alone will transfer heat to the cow.

Both low-pressure sprinklers and high-pressure misters mounted on fans can be effective if controlled correctly. However, high-pressure misters require an additional pump, are more prone to clogging, and need additional control. For these reasons, low-pressure sprinklers are more common.

Low-pressure sprinklers should be installed in the holding area and above the feedline but should not be installed above the stalls as constantly wet stalls can lead to a number of health issues. Install a half-circle (180-degree or less) nozzle every 6 to 8 feet along the feed line with each delivering 0.33 gallons per cow per cycle. 360-degree sprinklers should apply around 0.025 gallons per sq. ft. per cycle in the holding area.<sup>[9]</sup> Use pressure-controlled nozzles to limit system drain-down (waste) and increase the system's responsiveness. Sprinklers should be operated in a similar manner to table 3, although setpoints often also depend on available water and lagoon space.

Table 3. Recommended sprinkler setpoints.

Temperature (°F)	On Time (min.)	Cycle Time (min.)
68-78	1	15
79-88	1	10
>88	1	5

## **Mechanically Ventilated Barns**

### ***Tunnel or cross?***

Cross ventilation has become the preferred system in newly constructed barns housing large herds in the Midwest because it seems to provide better air velocity at cow level than in tunnel-ventilated barns. Baffles are more effective in cross-ventilated barns than in tunnel-ventilated barns because they are placed above the freestall rows (as opposed to perpendicular to the rows), and thus can more effectively direct air to cow level. Cross ventilation is also advantageous because (unlike in tunnel-ventilated barns) most of the cows in a cross-ventilated barn will be oriented parallel to the airflow and thus the upstream cows block less air from reaching the downstream cows than if the cows are oriented perpendicular to the airflow.

Despite those benefits, cross ventilation is not common in hot and humid climates. Tunnel-ventilated barns have the advantage of being convertible to natural ventilation in the summer. In addition, because naturally ventilated barns are usually 6 rows or less (and therefore significantly longer than they are wide), it requires more fans to retrofit a naturally ventilated barn for cross ventilation than tunnel. Typically, cross-ventilated barns have a minimum of 8 rows.

### ***Evaporative cooling in mechanically ventilated barns***

Mechanical ventilation is compatible with low-pressure sprinklers, high-pressure misters, and evaporative cooling pads. While evaporative cooling is less effective than in arid climates, it can still provide a significant cooling effect during the hottest hours of the day (when the RH is lowest). For example, at 90 °F and 60% RH (typical summer conditions in Florida), a cooling pad or misting system operating at 75% evaporation efficiency produces a 8.6 °F temperature reduction, which is equivalent to a reduction in THI of 3 units. Cooling pads cause an increase in static pressure, while misting requires more careful control because the flow rate must be adjusted to suit changing conditions. Some mechanically ventilated facilities in humid subtropical climates now rely on both evaporative cooling pads or high pressure misters to cool the air and low-pressure sprinklers to further cool the cows at the feedline.

### ***Baffles***

Baffles provide benefits in cross-ventilated barns when oriented directly over a row of head-to-head stalls. By reducing a barn's cross-sectional area, baffles increase the velocity of the air passing beneath them. As a result, the number of fans needed to produce a given velocity can be reduced.

However, because the air velocity is redistributed between the baffles, it cannot be said with certainty that they are worthwhile in tunnel-ventilated barns (where they only provide higher velocity air to the few stalls that they directly overhang). Some owners of tunnel-ventilated barns have opted to install (instead of baffles) full ceilings at a lower level. This measure, although more expensive, increases air velocity throughout the entire barn as opposed to only beneath the baffles. It should be noted that baffles installed above the alleys should be placed high enough (12 to 13 feet) so that equipment can pass beneath, but they can be placed at a lower height (6 to 10 feet) above the stalls.<sup>[7]</sup>

### **Fans for mechanical ventilation**

#### *Type*

Mechanically-ventilated barns are most commonly ventilated under negative pressure using exhaust fans. Positive pressure fans are used in barns that have significant infiltration, as well as in the milking parlor. In general, larger fans operate more efficiently than smaller fans. However, this is only true to a point. Independently tested 72" fans show dramatically reduced performance at higher static pressures.<sup>[10]</sup> Since fan capacity decreases as static pressure increases, fans should be chosen based on how they operate at the barn's estimated static pressure (see the companion summary article<sup>[5]</sup> for more information on estimating the static pressure of a specific barn).

#### *Quantity*

The total ventilation capacity chosen should meet two criteria: air exchange and air velocity at cow level. Summer air exchange rates should at a minimum be 1,000 CFM per cow. However, this value is usually significantly exceeded in order to attain the desired air velocity of 433 to 704 fpm at cow level.<sup>[13]</sup> To achieve these velocities at cow level in tunnel-ventilated barns, barns are sometimes designed with average air velocities of 704 to 880 fpm.<sup>[7]</sup> To determine the number of fans needed to provide the desired flow rate (also see example 1 in appendix), use the following steps:

1. Calculate cross-sectional area under baffle.  
 $\text{Area}_{\text{cs}} [\text{sq. ft.}] = \text{height}_{\text{baffle}} [\text{ft.}] \times \text{length}_{\text{baffle}} [\text{ft.}]$
2. Calculate airflow (Q) necessary to achieve design velocity.  
 $Q [\text{CFM}] = \text{Velocity} [\text{fpm}] \times \text{Area}_{\text{cs}} [\text{sq. ft.}]$
3. Calculate airflow necessary to achieve 1,000 CFM per cow.  
 $Q [\text{CFM}] = 1,000 [\text{CFM}/\text{cow}] \times \text{number of cows}$
4. Choose the larger of two airflow rates.
5. Divide larger flow rate by the airflow per fan at the barn's estimated static pressure.  
 $\text{Number of exhaust fans} = Q [\text{CFM}] / \text{Airflow per fan} [\text{CFM}/\text{fan}]$

#### *Controls*

Mechanical ventilation systems are staged so that some fans will always be on to provide the minimum ventilation rate, and as the temperature increases, additional fans will begin to run until all the fans are on at a temperature of 71-74 °F.<sup>[14]</sup> Many tunnel-ventilated barns are converted to natural ventilation during cooler weather. But if they are not, some ventilation fans must operate at all times.

While a minimum ventilation rate of 50 CFM/cow is recommended in colder climates, the mild weather ventilation rate of 170 CFM/cow<sup>[9]</sup> may be more suitable for subtropical winters. Exhaust fans equipped with VFDs can be purchased, but the airflow through the barn can also be adjusted by simply turning off some fans. VFD fans tend to be more expensive than single-speed fans, but also have the potential to operate more efficiently when only a portion of the total fan capacity is needed.

### ***Inlets***

A system's air inlets determine the points at which air is introduced to the barn and how that air is distributed. Generally, the inlet is located on the side of the barn opposite the exhaust fans. To function properly, a mechanical ventilation system must have properly sized inlets. Smaller inlets result in higher static pressure. Airflow through evaporative pads should be limited to 300 to 400 CFM/sq. ft. to limit static pressure, but the airflow can be as high as 600 to 800 CFM/sq. ft. if no pads are used.<sup>[7]</sup>

### **Estimating Resource Use**

The amount of electricity used by the fans can be estimated by multiplying the power consumption of the fans' motors by the typical number of hours per year the fans operate (see example 2 in appendix). When estimating electrical costs, note that motor hp is given in mechanical output and that the electrical input is always greater. A typical 1 hp motor requires around 1 kW of electrical power.<sup>[15]</sup>

The water used by a sprinkler system can be estimated in a similar manner. Simply multiply the system's setpoints by the typical number of hours per year within each setpoint range (see example 3 in appendix).

### **Conclusions**

Implementing fans and some form of evaporative cooling has been shown to provide clear economic benefits in humid subtropical climates. However, determining the practices that will optimize the combination of air velocity and evaporative cooling in terms of milk production, animal health, and resource efficiency will require further study. Although there are gaps in the current body of research, the guidelines contained herein should help to reduce the negative effects of heat stress in humid subtropical climates.

### **Acknowledgements**

We would like to thank Schaefer Ventilation Equipment of Sauk Rapids, MN, for their financial and technical assistance with this project. We want to especially thank Neil Crocker, Jim Evans, James Kleinke, and Pete Lyle. This material is also based upon work supported by the National Science Foundation Graduate Research Fellowship Program under grant No. DGE-1256259.

## Appendix:

### Example 1: Tunnel-ventilated barn

#### Assumptions:

- 6-row, 800-cow freestall
  - Dimensions of 120 ft. wide by 500 ft. long
  - Eave height of 12 ft. and ridge height of 22 ft.
  - Design velocity of 704 fpm
  - Assume 0.1 inches of static pressure for barn
  - Performance test results show exhaust fan moving 31,000 CFM at 0.10 inches of static pressure
- 
- **Calculate cross-sectional area:**
    - $A_{cs} = 12 \text{ ft.} * 120 \text{ ft.} + ((22 - 12) \text{ ft} * 120 \text{ ft})/2 = 2,040 \text{ sq. ft.}$
  - **Calculate volumetric flow rate to meet design velocity and air exchange per cow:**
    - Calculate airflow based on velocity:  $Q = 2,040 \text{ sq. ft.} * 704 \text{ fpm} = 1,436,160 \text{ CFM}$
    - Calculate airflow based on number of cows:  $Q = 800 \text{ cows} * 1,000 \text{ CFM/cow} = 800,000 \text{ CFM}$
    - Choose larger: 1,436,160 CFM
  - **Consider fans needed:**
    - Number of fans =  $1,436,160 \text{ CFM} / 31,000 \text{ CFM/fan} = 46 \text{ fans}$

### Example 2: Estimating the annual electrical cost of operating circulation fans in Gainesville, FL

#### Assumptions:

- 40 circulation fans with 1 hp motors in naturally ventilated barn
- 1 hp motor consumes 1 kW of power<sup>[15]</sup>
- Electricity costs \$0.11/kWh
- Set point temperature is 68 °F
- In typical meteorological year, Gainesville has 5,002 hours with a temperature at or above 68 °F<sup>[2]</sup>

(note: from Table 1, with a THI set point of 68, fans would only run 4,751 hours)

- Annual electrical costs [\$] = (hours over set point [hours/yr]) \* (number of fans) \*(electrical use per fan [kW/fan]) \*(costs of electricity [\$/kWh]) = 5,002 hours/year \* 40 fans \*1 kW/fan \* \$0.11/kWh = \$22,008/year

**Example 3: Estimating the annual water use of a sprinkler system operating in a holding area in Gainesville, Florida**

**Assumptions:**

- Holding area is 400 square feet
  - Water is applied at a rate of 0.025 gal per sq. ft. per cycle
  - System uses valved nozzles, so water is not wasted between cycles
  - Cycles are set to operate 1 minute in: 15 minutes at temperature 68 – 77 °F, 10 minute cycles at temperatures 78 – 88 °F, and 5 minute cycles at temperatures above 88 °F
  - Typical meteorological year has 3,240 hours from 68 – 78 °F, 1394 hours from 79 – 88 °F, and 368 hours > 88 °F
- Water per cycle [gal] = square feet \* application per square foot = 400 square feet \* 0.025 gal per square foot per cycle = 10 gal per cycle
- Number of cycles is found by dividing the number of hours by the cycle duration

Range	Number of hours/yr	Number of cycles/yr	Gallons of water/yr
68 – 78 °F	3,240	12,960	129,600
79 – 88 °F	1,394	8,364	83,640
> 88 °F	368	4,416	44,160
<b>Annual water use in holding area:</b>			<b>257,400</b>

(note: water can be saved in the holding area by staging sprinklers so they do not operate in the empty portion (or only installing sprinklers in the 75% closest to the parlor, which is occupied a higher percentage of the time). Significant water could also be saved by staging the sprinklers to turn on at THI setpoints instead of temperature setpoints).

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